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Brockley et al.

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(54) **VEHICLE SPEED CONTROL SYSTEM**

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See application file for complete search history.

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2710/105 (2013.01)

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2050/0052; B60W 2050/0056; B60W
2540/10; B60W 2710/0666; B60W 2710/105

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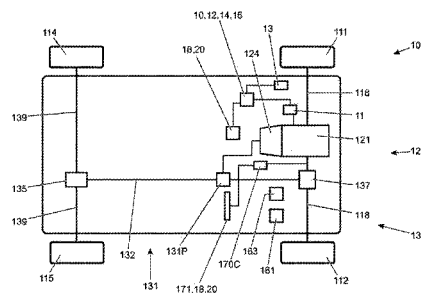
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(57) **ABSTRACT**

A control system for a vehicle operable to implement a
speed control function, the control system being operable to:
receive an input of a target speed at which the vehicle is
intended to travel; determine an instantaneous value of a
primary target speed torque parameter corresponding to an
amount of torque that should be developed at a given
position in a powertrain in order to control the vehicle to
travel at the target speed; and filter by means of filter means
a value of a torque parameter input thereto to generate a
filtered torque parameter value, the system being operable to
command the powertrain to develop at said position an
amount of torque corresponding to the filtered torque param-
eter value, wherein the system is operable to input to the
filter means a secondary target speed torque parameter value
determined in dependence on the primary target speed
torque parameter value and a current value of filtered torque
parameter generated by the filter means.

14 Claims, 9 Drawing Sheets



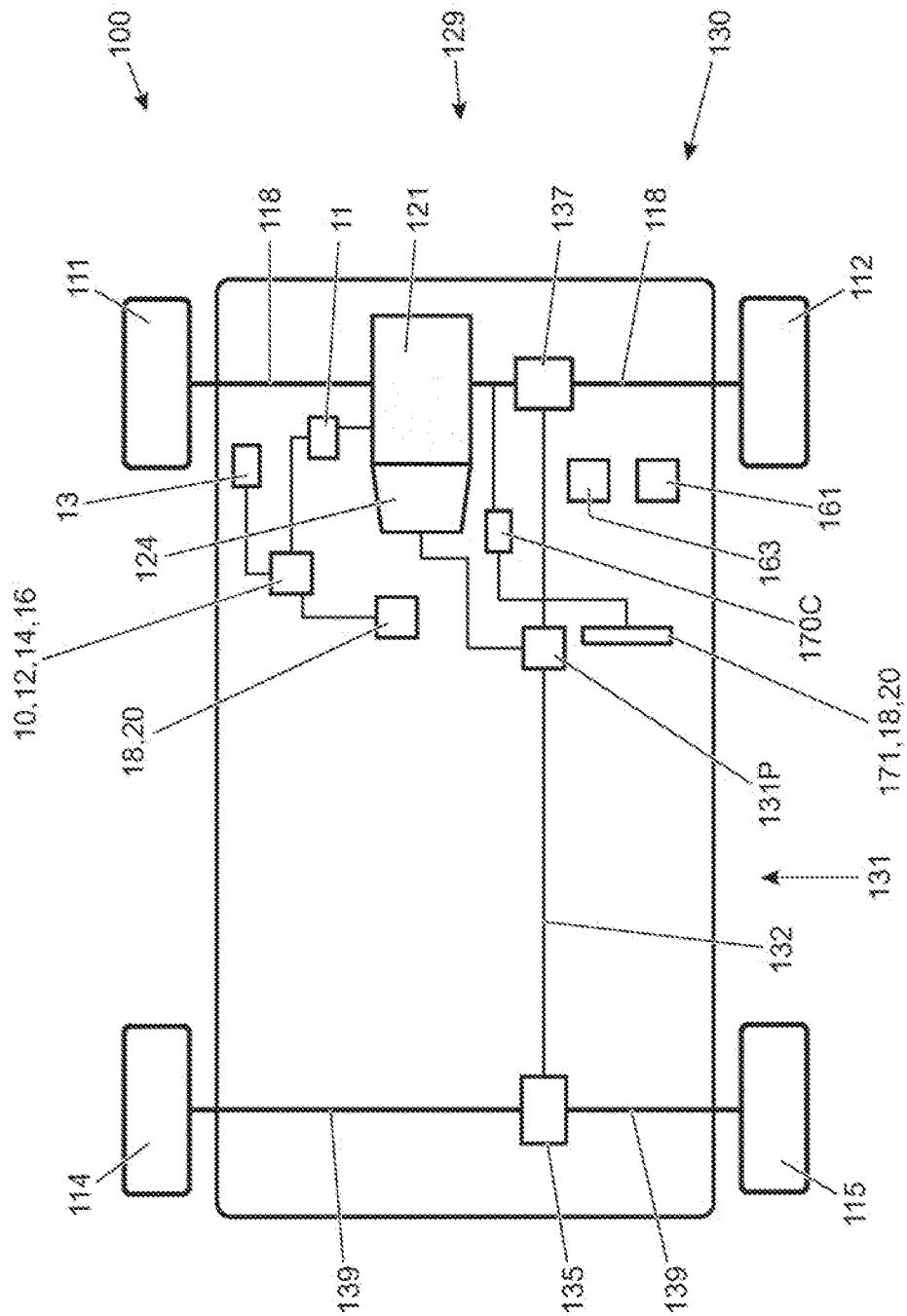


FIGURE 1

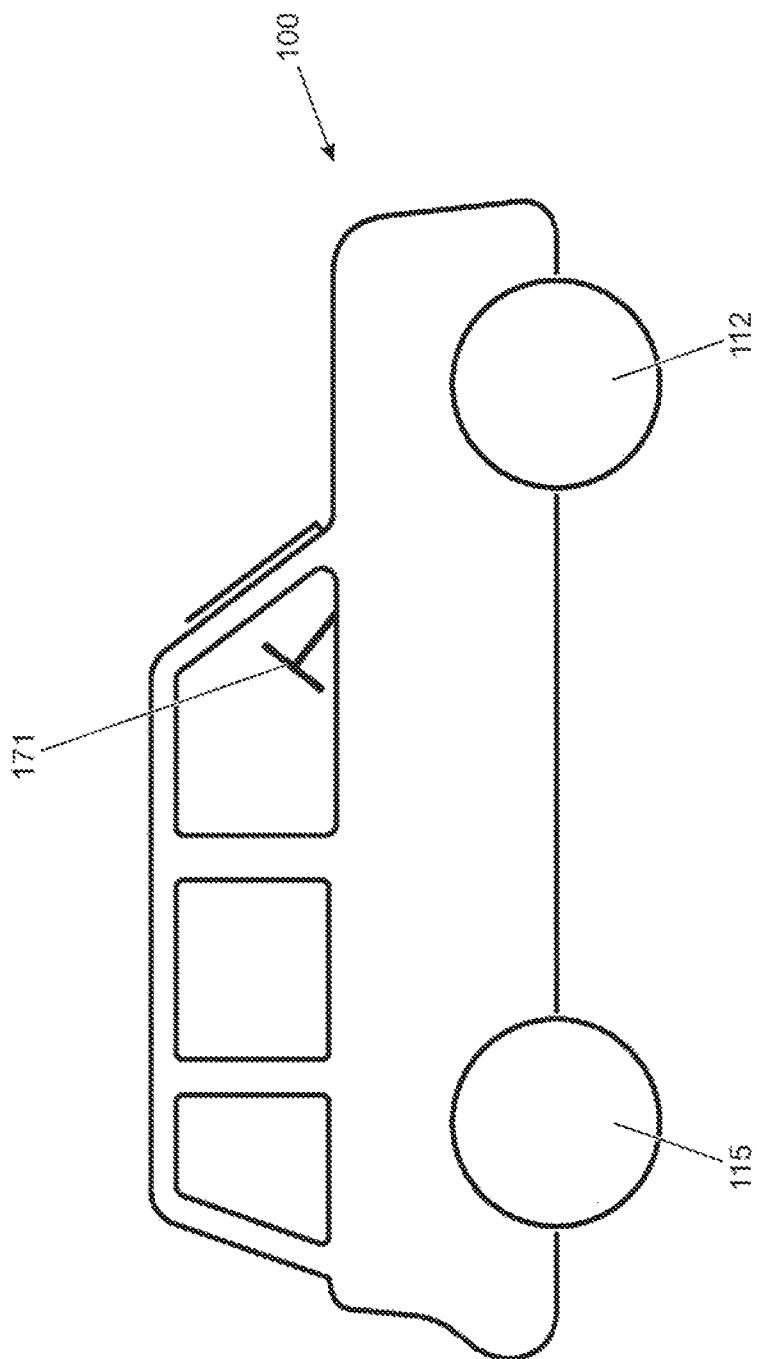


FIGURE 2

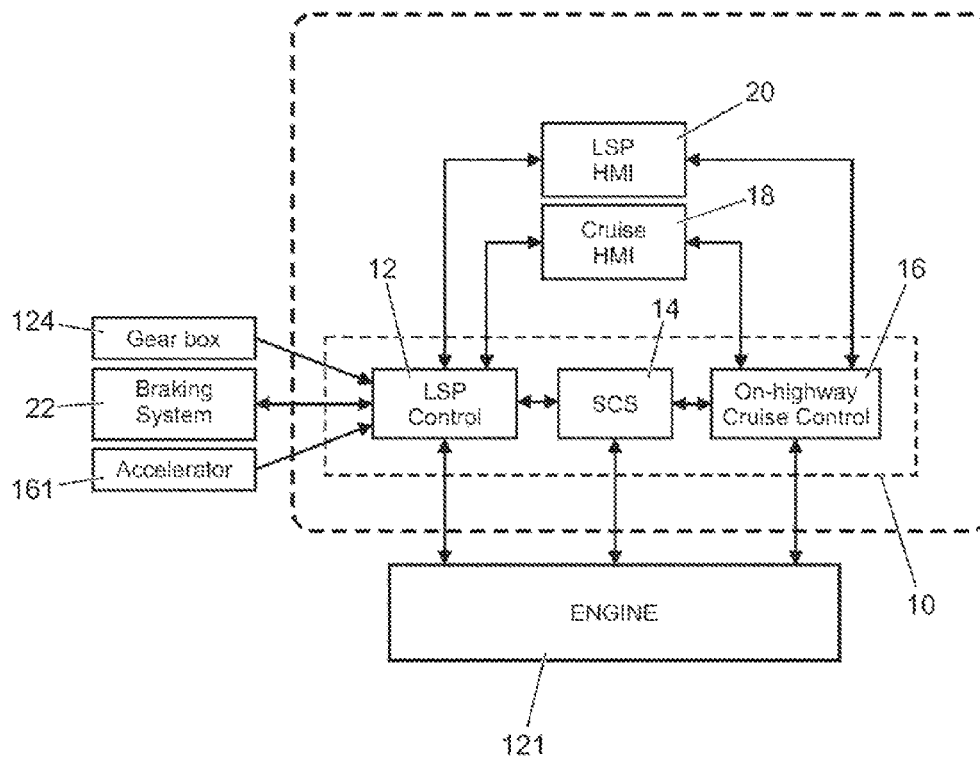


FIGURE 3

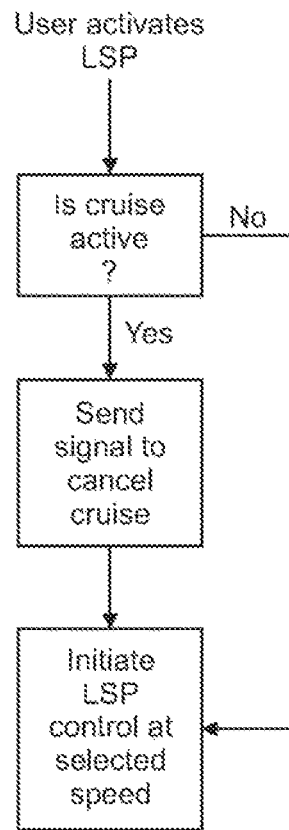


FIGURE 4

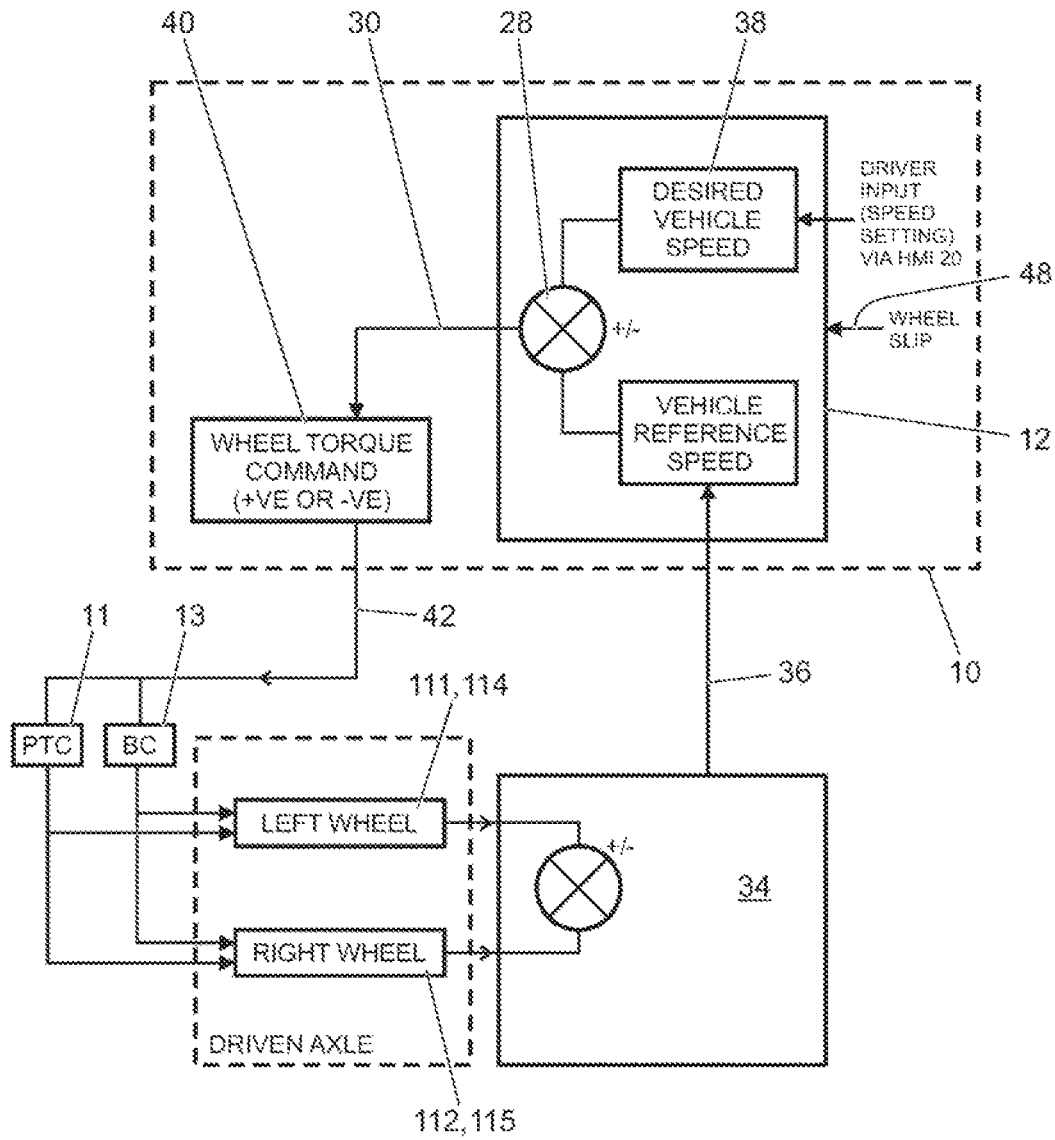


FIGURE 5

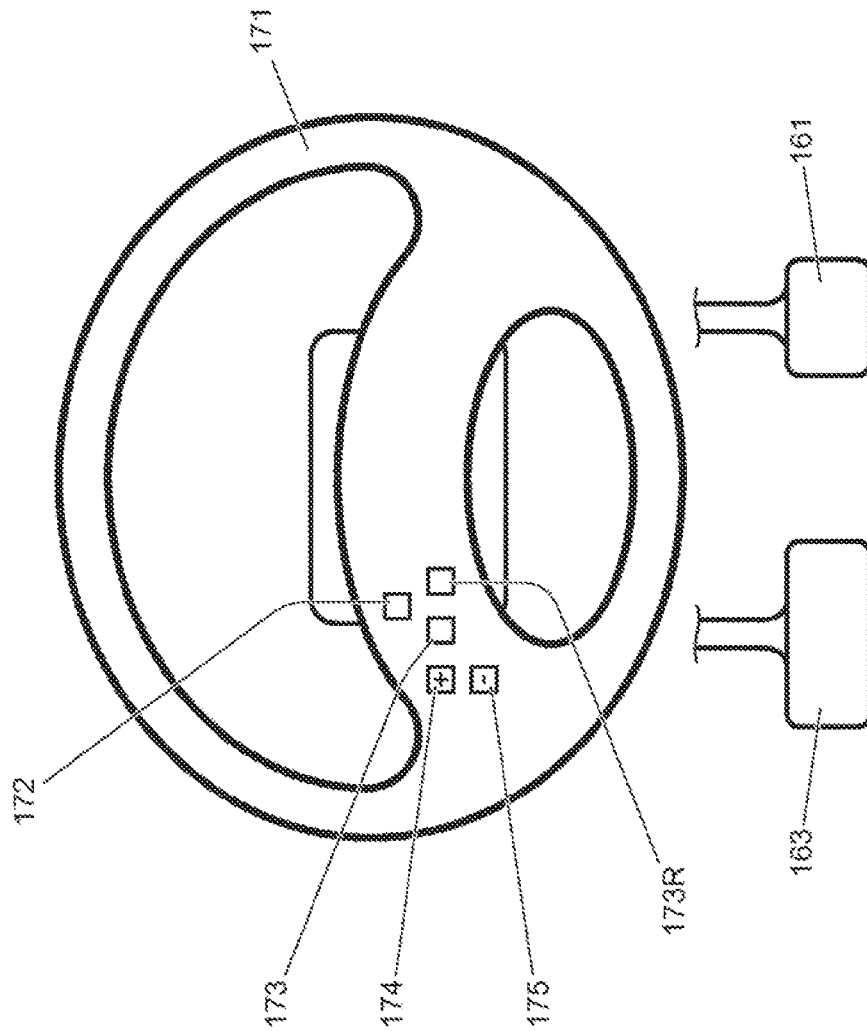


FIGURE 6

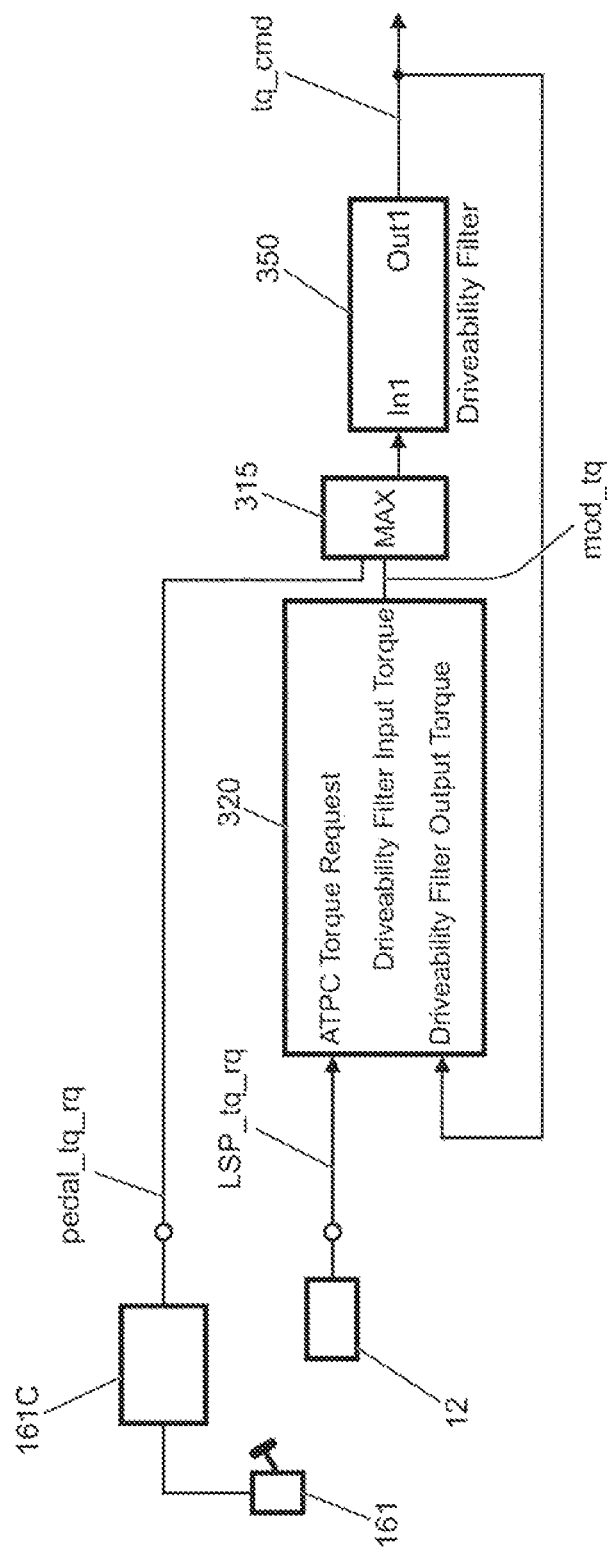


FIGURE 7

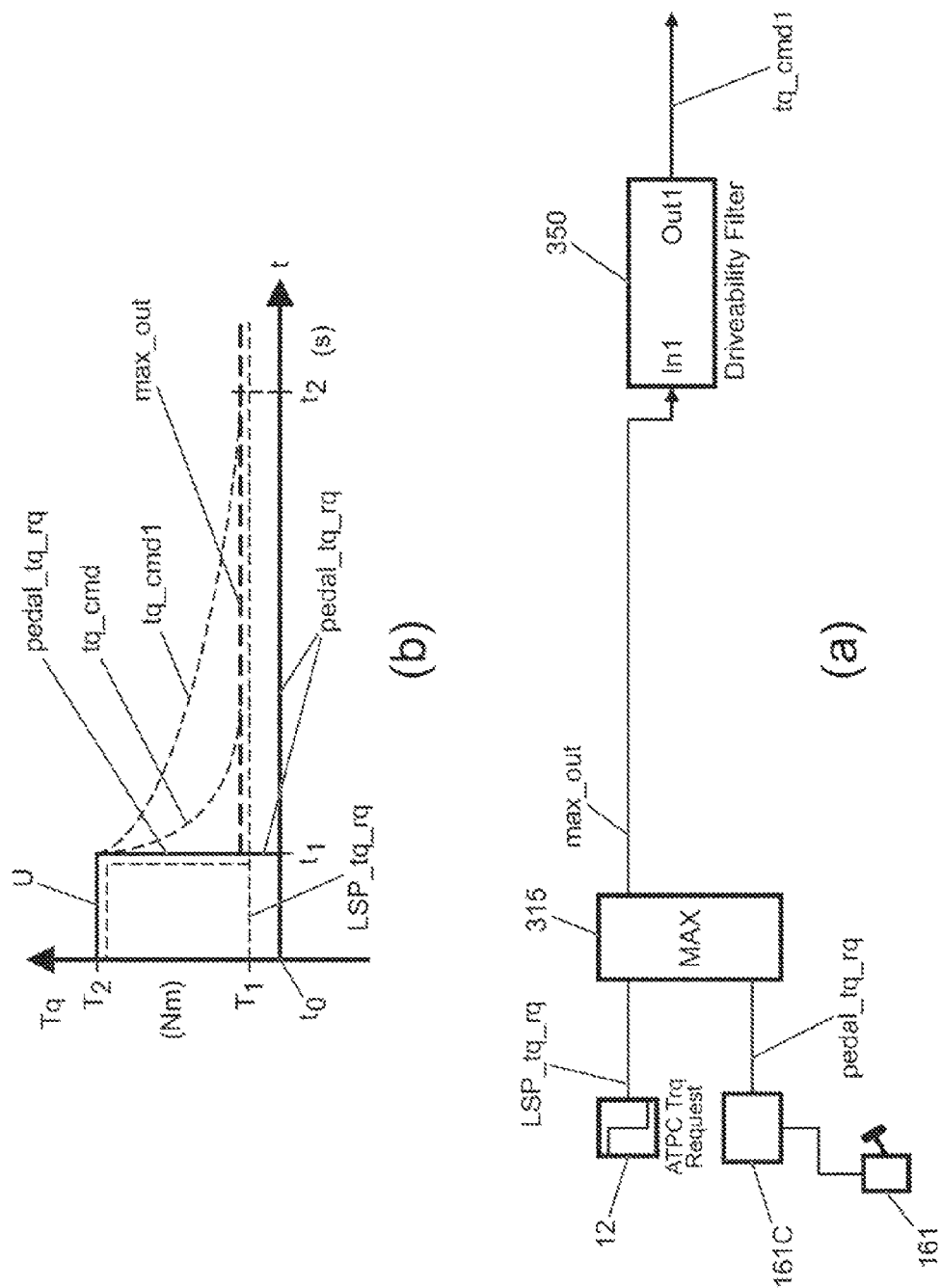


FIGURE 8

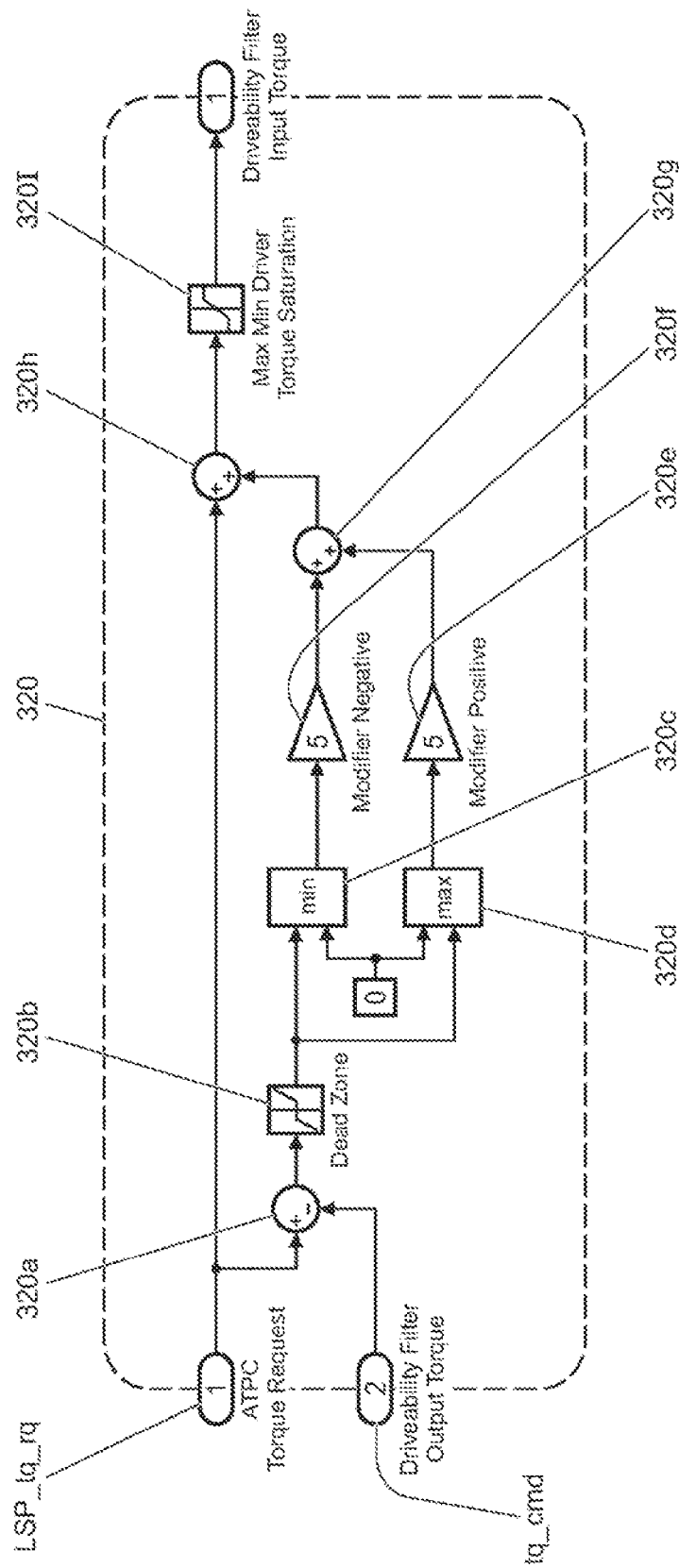


FIGURE 9

VEHICLE SPEED CONTROL SYSTEM**FIELD OF THE INVENTION**

The invention relates to a system for controlling the speed of a vehicle. In particular, but not exclusively, the invention relates to a system for controlling the speed of a land-based vehicle which is capable of driving in a variety of different and extreme terrains and conditions.

The content of co-pending UK patent application no GB1214651.0 is hereby incorporated by reference.

BACKGROUND TO THE INVENTION

In known vehicle speed control systems, typically referred to as cruise control systems, the vehicle speed is maintained on-road once set by the user without further intervention by the user so as to improve the driving experience for the user by reducing workload.

The user selects a speed at which the vehicle is to be maintained, and the vehicle is maintained at that speed for as long as the user does not apply a brake or, in some systems, the clutch. The cruise control system takes its speed signal from the driveshaft or wheel speed sensors. When the brake or the clutch is depressed, the cruise control system is disabled so that the user can change the vehicle speed without resistance from the system. If the user depresses the accelerator pedal the vehicle speed will increase, but once the user removes his foot from the accelerator pedal the vehicle reverts to the pre-set cruise speed.

More sophisticated cruise control systems are integrated into the engine management system and may include an adaptive functionality which takes into account the distance to the vehicle in front using a radar-based system. For example, the vehicle may be provided with a forward-looking radar detection system so that the speed and distance of the vehicle in front is detected and a safe following speed and distance is maintained automatically without the need for user input. If the lead vehicle slows down, or another object is detected by the radar detection system, the system sends a signal to the engine or the braking system to slow the vehicle down accordingly.

Such systems are usually operable only above a certain speed, typically around 15-20 mph, and are ideal in circumstances in which the vehicle is travelling in steady traffic conditions, and particularly on highways or motorways. In congested traffic conditions, however, where vehicle speed tends to vary widely, cruise control systems are ineffective, and especially where the systems are inoperable because of a minimum speed requirement. A minimum speed requirement is often imposed on cruise control systems so as to reduce the likelihood of low speed collision, for example when parking. Such systems are therefore ineffective in certain driving conditions (e.g. low speed) and are set to be automatically disabled in circumstances in which a user may not consider it to be desirable to do so.

Known cruise control systems also cancel in the event that a wheel slip event is detected requiring intervention by a traction control system (TCS) or stability control system (SCS). Accordingly, they are not well suited to maintaining vehicle progress when driving in off road conditions where such events may be relatively common.

It is also known to provide a control system for a motor vehicle for controlling one or more vehicle subsystems. U.S. Pat. No. 7,349,776, the content of which is hereby incorporated by reference, discloses a vehicle control system comprising a plurality of subsystem controllers including an

engine management system, a transmission controller, a steering controller, a brakes controller and a suspension controller. The subsystem controllers are each operable in a plurality of subsystem function modes. The subsystem controllers are connected to a vehicle mode controller which controls the subsystem controllers to assume a required function mode so as to provide a number of driving modes for the vehicle. Each of the driving modes corresponds to a particular driving condition or set of driving conditions, and in each mode each of the sub-systems is set to the function mode most appropriate to those conditions. Such conditions are linked to types of terrain over which the vehicle may be driven such as grass/gravel/snow, mud and ruts, rock crawl, sand and a highway mode known as 'special programs off' (SPO). The vehicle mode controller may be referred to as a Terrain Response (TR) (RTM) System or controller.

It is desirable to provide a speed control system capable of controlling vehicle speed at relatively low speeds and whilst driving in off road conditions.

STATEMENTS OF INVENTION

Embodiments of the invention may be understood with reference to the appended claims.

Aspects of the present invention provide a system, a vehicle and a method.

In one aspect of the invention for which protection is sought there is provided a control system for a vehicle operable to implement a speed control function, the control system being operable to:

receive an input of a target speed at which the vehicle is intended to travel;

determine an instantaneous value of a primary target speed torque parameter corresponding to an amount of torque that should be developed at a given position in a powertrain in order to control the vehicle to travel at the target speed; and

filter by means of filter means a value of a torque parameter input thereto to generate a filtered torque parameter value,

the system being operable to command the powertrain to develop at said position an amount of torque corresponding to the filtered torque parameter value,

wherein the system is operable to input to the filter means a secondary target speed torque parameter value determined in dependence on the primary target speed torque parameter value and a current value of filtered torque parameter generated by the filter means.

The primary target speed torque parameter may have a value substantially equal to the amount of torque required to control the vehicle to travel at the target speed. Alternatively the primary target speed torque parameter may have a value indicative of the amount of torque required.

By given position of the powertrain is meant any suitable position of the powertrain, such as an engine output shaft for a vehicle having an engine, a gearbox or transmission input shaft, output shaft, drive shaft, one or more road wheels or any other suitable location of the powertrain.

The present inventors have recognised that in vehicles not according to an embodiment of the present invention, a substantial difference may be observed in respect of the responsiveness of a vehicle to changes in speed depending on whether speed control is active. This can result in a reduction of driver confidence in the speed control system.

Embodiments of the present invention have the advantage that a difference in a response of a vehicle to changes in powertrain torque demand in dependence on whether or not

the control system is implementing the speed control function may be reduced. This feature has the advantage that driver confidence in the speed control function may be enhanced.

Embodiments of the invention may have at least two advantages. The first is that, when speed control is performed by the control system, a responsiveness of the powertrain to changes in torque demanded by the speed control function may be enhanced, since any phase lag between the filtered torque parameter value and target speed torque parameter value may be reduced.

The second is that, if a driver over-rides speed control by demanding more torque than that corresponding to the target speed torque parameter value, for example by depressing an accelerator pedal or the like by a sufficient amount, when the driver subsequently stops over-riding speed control the filtered torque parameter value falls to the primary target speed torque parameter value much more quickly.

For example, if a driver is travelling without speed control active at 10 kph and wishes to reduce vehicle speed to 5 kph, the driver might temporarily release an accelerator control such as an accelerator pedal and allow vehicle speed to fall to a value close to the target speed. The driver might then actuate the accelerator pedal to maintain the target speed. Engine braking might therefore be provided by the powertrain to slow the vehicle during the period in which the accelerator pedal is in a released condition. In other words, negative torque may be developed by the powertrain to slow the vehicle, resulting in a more rapid decrease in vehicle speed.

In contrast, if speed control by the control system is active and the driver over-rides the speed control function by demanding more powertrain torque than that requested by the speed control function, for example in order to move more quickly past a particular location, and then releases the accelerator pedal and allows the control system to resume vehicle control, the amount of torque demanded of the powertrain whilst the vehicle decelerates may not fall towards a value corresponding to a released accelerator pedal position as quickly as in the case where speed control is not active. Accordingly the driver may perceive the vehicle speed to be 'floating', i.e. falling too slowly, and the vehicle relatively unresponsive to the release of the accelerator pedal by the driver.

Embodiments of the present invention overcome this problem by modifying the value of a torque parameter input to the filter means according to the difference between the primary target speed torque parameter and the filtered torque parameter value. This has the effect that the filtered torque value tends towards the primary target speed torque value more quickly.

Advantageously the system may be operable to determine the value of the secondary target speed torque parameter in dependence on a difference between the value of the primary target speed torque parameter and the current value of the filtered torque parameter.

The system may be operable wherein if the value of the primary target speed torque parameter is less than a current value of the filtered torque parameter, the value of the secondary target speed torque parameter is set to a value that is less than the primary target speed torque parameter by an amount that is determined in dependence on the primary target speed torque parameter value and the current value of filtered torque parameter.

This feature has the advantage that a rate at which the amount of torque developed by the powertrain decreases to the primary target speed torque may be made to correspond

more closely to that which would be experienced in the event a driver were to slow the vehicle to the target speed without using the control system speed control functionality.

The amount by which the value of torque parameter applied to the filter means is less than the target speed torque parameter may be an amount substantially equal to the difference between the value of target speed torque parameter and the current value of filtered torque parameter. Alternatively the amount may be a multiple of the difference. The multiple may be less than 1 or greater than one, such as 0.5, 1.5, 2, 3, 4, 5 or any other suitable value. Thus the amount may be non-integer. Other arrangements are also useful.

The system may be operable to input to the filter means a value of secondary target speed torque parameter that is less than the primary target speed torque parameter by an amount corresponding to the difference between the filtered torque parameter value and the primary target speed torque parameter value.

The system may be operable to input to the filter means a value of secondary target speed torque parameter that is less than the primary target speed torque parameter value by an amount substantially equal to the difference between the filtered torque parameter value and the primary target speed torque parameter value, multiplied by a factor.

The factor may be less than or greater than 1.

The system may be operable such that if the value of primary target speed torque parameter is greater than that of the filtered torque parameter the value of secondary target speed torque parameter input to the filter means is increased to a value above that of the primary target speed torque parameter value, the amount of the increase being determined in dependence on the primary target speed torque parameter value and the current value of filtered torque parameter.

The system may be operable to input to the filter means a value of secondary target speed torque parameter that is greater than the primary target speed torque parameter value by an amount substantially equal to the difference between the filtered torque parameter value and the primary target speed torque parameter value.

The system may be operable to input to the filter means a value of secondary target speed torque parameter that is greater than the primary target speed torque parameter value by an amount substantially equal to the difference between the filtered torque parameter value and the primary target speed torque parameter value, multiplied by a factor.

The system may be operable to limit an upper and lower value of secondary target speed torque parameter input to the filter means to respective upper and lower saturation values.

This feature has the advantage that extreme excursions in the value of powertrain torque parameter when under speed control may be prevented. Thus, whilst it has been stated above that the modifier means is arranged such that if the value of target speed torque parameter is less than a current value of filtered torque parameter, the modifier means reduces the value of torque parameter output thereby to the filter means, the modifier means will not reduce the value of torque parameter output thereby to the filter means below a prescribed value. Similarly, the modifier means will not increase the value of torque parameter output thereby to a value that is above the upper saturation value.

The system may be operable to input to the filter means the higher of the secondary target speed torque parameter value and a value of a driver demanded torque parameter corresponding to a setting of a user-operable accelerator control.

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Thus, the same filter may be used to filter torque values demanded by the control system and by a driver.

This feature eliminates the requirement to provide a control system that takes into account functions associated with the filter means of the particular vehicle in which the control system is installed. Thus the same control system may be installed in a range of different vehicles each having filter means with different respective characteristics.

It is to be understood that the filter means is typically employed in vehicles regardless of whether a speed control function is provided and is arranged to mitigate the effects of driveline or powertrain shunt when a user varies powertrain torque demand (e.g. by varying accelerator pedal position). By employing the filter means when speed control is in progress, a workload on the designer of the speed control function is reduced since the factors taken into account by the filter means do not need to be taken into account by the speed control function to the same extent, if at all. Thus, issues such as mitigation of powertrain shunt may be taken into account by the filter means allowing similar or substantially identical speed control functions to be employed in a range of different vehicles having powertrains with different characteristics such as stiffness and resonant frequencies.

The filter means may be arranged to subject the value of torque parameter input thereto to a low pass filter.

The system may be operable to receive a user input of the target speed.

In a further aspect of the invention for which protection is sought there is provided a vehicle comprising a control system according to a preceding aspect.

Embodiments of the present invention are useful in a range of different types of vehicles including vehicles having a single engine such as a single internal combustion engine as the source of propulsion, in hybrid electric vehicles and in electric vehicles.

In one aspect of the invention for which protection is sought there is provided a method of controlling a speed of a vehicle comprising:

receiving an input of a target speed at which the vehicle is intended to travel; determining an instantaneous value of a primary target speed torque parameter corresponding to an amount of torque that should be developed at a given position in a powertrain in order to control the vehicle to travel at the target speed; filtering by filter means a value of a torque parameter input thereto to generate a filtered torque parameter value; and commanding the powertrain to develop at said position an amount of torque corresponding to the filtered torque parameter value, wherein filtering by the filter means the value of torque parameter input thereto comprises filtering a secondary target speed torque parameter value determined in dependence on the primary target speed torque parameter value and a current value of filtered torque parameter generated by the filter means.

In one aspect of the invention for which protection is sought there is provided a control system for a vehicle operable to implement a speed control function, the control system comprising:

means for receiving a user input of a target speed at which the vehicle is intended to travel; target speed torque determining means for determining an instantaneous value of torque, target speed torque, that should be applied to one or more wheels of the vehicle by a powertrain in order to control the vehicle to travel at the target speed; and

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filter means operable to filter a value of torque input thereto to generate a filtered torque value,

the system being operable to command the powertrain to apply to the one or more wheels an amount of torque corresponding to the filtered torque value,

wherein the system further comprises modifier means operable to receive the instantaneous value of target speed torque generated by the target speed torque determining means and to input to the filter means a value of torque determined in dependence on a difference between the target speed torque value and a current value of filtered torque.

In one aspect of the invention for which protection is sought there is provided a control system for a vehicle operable to implement a speed control function, the control system comprising:

means for receiving a user input of a target speed at which the vehicle is intended to travel;

target speed torque parameter determining means for determining an instantaneous value of a target speed torque parameter corresponding to an amount of torque that should be developed at a given position in a powertrain in order to control the vehicle to travel at the target speed; and

filter means operable to filter a value of a torque parameter input thereto to generate a filtered torque parameter value,

the system being operable to command the powertrain to develop at said position an amount of torque corresponding to the filtered torque parameter value,

wherein the system further comprises modifier means operable to receive the instantaneous value of target speed torque parameter generated by the target speed torque parameter determining means and to input to the filter means a value determined in dependence on the target speed torque parameter value and a current value of filtered torque parameter generated by the filter means.

Advantageously the modifier means is operable to input to the filter means a value of torque parameter determined in dependence on a difference between the target speed torque parameter value and the current value of filtered torque parameter.

The modifier means may be arranged such that if the value of the target speed torque parameter is less than a current value of the filtered torque parameter, the modifier means reduces the value of torque parameter output thereby to the filter means to a value below that of the target speed torque parameter, the amount of the reduction being determined in dependence on the target speed torque parameter value and the current value of filtered torque parameter.

The system may be operable to apply to the filter means a value of torque parameter that is less than the target speed torque parameter value by an amount substantially equal to the difference between the substantially instant filtered torque parameter value and the substantially instant target speed torque parameter value.

Advantageously the modifier means may be arranged such that if the value of target speed torque parameter is greater than that of the filtered torque parameter the modifier means increases the value of torque parameter output thereby to the filter means to a value above that of the target speed torque parameter value, the amount of the increase being determined in dependence on the target speed torque parameter value and the current value of filtered torque parameter.

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The system may be operable to apply to the filter means a value of powertrain torque parameter that is greater than the target speed torque parameter value by an amount substantially equal to the difference between the substantially instant filtered torque parameter value and the substantially instant target speed torque parameter value.

The modifier means may be operable to limit an upper and lower value of torque parameter output thereby to respective upper and lower saturation values.

Advantageously the system may comprise means for generating a value of driver demanded torque parameter in dependence on a position of a user operable accelerator control, the system being operable to apply to the filter means the higher of driver demanded torque parameter value and the value of torque parameter output by the modifier means.

The filter means may be arranged to subject the value of torque parameter input thereto to a low pass filter.

In a further aspect of the invention for which protection is sought there is provided a vehicle comprising a control system according to a preceding aspect.

In another aspect of the invention for which protection is sought there is provided a method of controlling a speed of a vehicle comprising:

receiving a user input of a target speed at which the vehicle is intended to travel;

determining an instantaneous value of a target speed torque parameter corresponding to an amount of torque that should be developed at a given position in a powertrain in order to control the vehicle to travel at the target speed;

filtering by filter means a value of a torque parameter input thereto to generate a filtered torque parameter value; and

commanding the powertrain to develop at said position an amount of torque corresponding to the filtered torque parameter value,

wherein filtering by the filter means the value of torque parameter input thereto comprises filtering a torque parameter value determined in dependence on the target speed torque parameter value and a current value of filtered torque parameter generated by the filter means.

It is to be understood that the target speed may also be referred to as a 'set-speed' and the terms 'target speed' and 'set-speed' are used interchangeably herein.

In one aspect of the invention there is provided a speed control system modifier operable to receive from a speed control system a speed control signal corresponding to a required amount of torque and a signal from a torque filter corresponding to a filtered torque value, the modifier being operable to output to the torque filter a torque signal that is determined in dependence on the value of speed control signal and filtered torque value. The signal may be determined by subtracting from the speed control signal a value corresponding to the difference between the speed control signal and the filtered torque value. The signal may be substantially equal to this difference multiplied by a factor.

It will be appreciated that preferred and/or optional features of any one aspect of the invention may be incorporated alone or in appropriate combination within the any other aspect of the invention also.

BRIEF DESCRIPTION OF THE DRAWINGS

The invention will now be described by way of example only with reference to the following figures in which:

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FIG. 1 is a schematic illustration of a vehicle according to an embodiment of the invention in plan view;

FIG. 2 shows the vehicle of FIG. 1 in side view;

FIG. 3 is a high level schematic diagram of an embodiment of the vehicle speed control system of the present invention, including a cruise control system and a low-speed progress control system;

FIG. 4 is a flow diagram to illustrate the interaction between the cruise control system and the low-speed progress control system in FIG. 3;

FIG. 5 is a schematic diagram of further features of the vehicle speed control system in FIG. 3;

FIG. 6 illustrates a steering wheel and brake and accelerator pedals of a vehicle according to an embodiment of the present invention;

FIG. 7 shows a portion of a vehicle speed control system according to an embodiment of the present invention;

FIG. 8 shows (a) a portion of a vehicle speed control system not being a system according to an embodiment of the present invention and (b) a plot of filtered powertrain torque as a function of time following driver intervention; and

FIG. 9 shows a portion of a vehicle speed control system according to an embodiment of the present invention in more detail.

DETAILED DESCRIPTION

References herein to a block such as a function block are to be understood to include reference to software code for performing the function or action specified in which an output is provided responsive to one or more inputs. The code may be in the form of a software routine or function called by a main computer program, or may be code forming part of a flow of code not being a separate routine or function. Reference to function block is made for ease of explanation of the manner of operation of the controller.

FIG. 1 shows a vehicle 100 according to an embodiment of the invention having a powertrain 129. The powertrain 129 includes an engine 121 that is connected to a driveline 130 having an automatic transmission 124. Embodiments of the present invention are suitable for use in vehicles with manual transmissions, continuously variable transmissions or any other suitable transmission.

The driveline 130 is arranged to drive a pair of front vehicle wheels 111, 112 by means of a front differential 137 and a pair of front drive shafts 118. The driveline 130 also comprises an auxiliary driveline portion 131 arranged to drive a pair of rear wheels 114, 115 by means of an auxiliary driveshaft or prop-shaft 132, a rear differential 135 and a pair of rear driveshafts 139. Embodiments of the invention are suitable for use with vehicles in which the transmission is arranged to drive only a pair of front wheels or only a pair of rear wheels (i.e. front wheel drive vehicles or rear wheel drive vehicles) or selectable two wheel drive/four wheel drive vehicles. In the embodiment of FIG. 1 the transmission 124 is releasably connectable to the auxiliary driveline portion 131 by means of a power transfer unit (PTU) 131P, allowing selectable two wheel drive or four wheel drive operation. It is to be understood that embodiments of the invention may be suitable for vehicles having more than four wheels or where only two wheels are driven, for example two wheels of a three wheeled vehicle or four wheeled vehicle or a vehicle with more than four wheels.

A control system for the vehicle engine 121 includes a central controller, referred to as a vehicle control unit (VCU) 10, a powertrain controller 11, a brake controller 13 and a

steering controller **170C**. The brake controller **13** forms part of a braking system **22** (FIG. 3). The VCU **10** receives and outputs a plurality of signals to and from various sensors and subsystems (not shown) provided on the vehicle. The VCU **10** includes a low-speed progress (LSP) control system **12** shown in FIG. 3 and a stability control system (SCS) **14**, the latter being a known component of existing vehicle control systems. The SCS **14** improves the safety of the vehicle **100** by detecting and managing loss of traction. When a reduction in traction or steering control is detected, the SCS **14** is operable automatically to command a brake controller **13** to apply one or more brakes of the vehicle to help to steer the vehicle **100** in the direction the user wishes to travel. In the embodiment shown the SCS **14** is implemented by the VCU **10**. In some alternative embodiments the SCS **14** may be implemented by the brake controller **13**. Further alternatively, the SCS **14** may be implemented by a separate controller.

Although not shown in detail in FIG. 3, the VCU **10** further includes a Dynamic Stability Control (DSC) function block, a Traction Control (TC) function block, an Anti-Lock Braking System (ABS) function block and a Hill Descent Control (HDC) function block. These function blocks are implemented in software code run by a computing device of the VCU **10** and provide outputs indicative of, for example, DSC activity, TC activity, ABS activity, brake interventions on individual wheels and engine torque requests from the VCU **10** to the engine **121** in the event a wheel slip event occurs. Each of the aforementioned events indicate that a wheel slip event has occurred. Other vehicle sub-systems such as a roll stability control system or the like may also be useful.

The vehicle **100** also includes a cruise control system **16** which is operable to automatically maintain vehicle speed at a selected speed when the vehicle is travelling at speeds in excess of 30 kph. The cruise control system **16** is provided with a cruise control HMI (human machine interface) **18** by which means the user can input a target vehicle speed to the cruise control system **16** in a known manner. In one embodiment of the invention, cruise control system input controls are mounted to a steering wheel **171** (FIG. 6). Depression of a 'set-speed' control **173** sets the set-speed to the current vehicle speed. Depression of a '+' button **174** allows the set-speed to be increased whilst depression of a '-' button **175** allows the set-speed to be decreased. In some embodiments, if the cruise control system **16** is not active when the '+' button **174** is depressed, the cruise control system **16** is activated.

The cruise control system **16** monitors vehicle speed and any deviation from the target vehicle speed is adjusted automatically so that the vehicle speed is maintained at a substantially constant value, typically in excess of 30 kph. In other words, the cruise control system is ineffective at speeds lower than 30 kph. The cruise control HMI **18** may also be configured to provide an alert to the user about the status of the cruise control system **16** via a visual display of the HMI **18**.

The LSP control system **12** provides a speed-based control system for the user which enables the user to select a very low target speed at which the vehicle can progress without any pedal inputs being required by the user. This low-speed progress control function is not provided by the on-highway cruise control system **16** which operates only at speeds above 30 kph. Furthermore, known on-highway cruise control systems including the present system **16** are configured so that, in the event that the user depresses the brake or the clutch, the cruise control function is cancelled

and the vehicle **100** reverts to a manual mode of operation which requires user pedal input to maintain vehicle speed. In addition, detection of a wheel slip event, as may be initiated by a loss of traction, also has the effect of cancelling the cruise control function.

The LSP control system **12** is operable to apply selective powertrain, traction control and braking actions to the wheels of the vehicle, collectively or individually, to maintain the vehicle **100** at the desired speed. It is to be understood that if the vehicle **100** is operating in a two wheel drive mode in which only front wheels **111**, **112** are driven, the control system **12** may be prevented from applying drive torque to rear wheels **113**, **114** of the vehicle **100**.

The user inputs the desired target speed to the LSP control system **12** via a low-speed progress control HMI (LSP HMI) **20** (FIG. 1, FIG. 3). The LSP control system **12** operates at vehicle speeds typically below about 50 kph but does not activate until vehicle speed drops to below 30 kph when the cruise control system of the vehicle becomes ineffective. The LSP control system **12** is configured to operate independently of a traction event, i.e. the system **12** does not cancel speed control upon detection of wheel slip. Rather, the LSP control system **12** actively manages vehicle behaviour and in this way, at least, differs from the functionality of the cruise control system **16**, as will be described in further detail below.

The LSP control HMI **20** is provided in the vehicle cabin so as to be readily accessible to the user. The user of the vehicle is able to input to the LSP control system **12**, via the LSP HMI **20**, an indication of the speed at which the user desires the vehicle to travel (referred to as "the target speed"). The LSP HMI **20** also includes a visual display upon which information and guidance can be provided to the user about the status of the LSP control system **12**.

The LSP control system **12** receives an input from the braking system **22** of the vehicle indicative of the extent to which the user has applied braking by means of a brake pedal **163**. The LSP control system **12** also receives an input from an accelerator pedal **161** indicative of the extent to which the user has depressed the accelerator pedal **161**. An input is also provided to the LSP control system **12** from the transmission or gearbox **124**. This input may include signals representative of, for example, the speed of an output shaft of the gearbox **124**, torque converter slip and a gear ratio request. Other inputs to the LSP control system **12** include an input from the cruise control HMI **18** which is representative of the status (ON/OFF) of the cruise control system **16**, and an input from the LSP control HMI **20** which is representative of the status of the LSP control function.

The cruise control HMI **18** and the LSP HMI **20** have input controls provided on a steering wheel of the vehicle for convenience of operation by the user.

FIG. 6 shows the steering wheel **171** of the vehicle **100** of FIG. 1 in more detail, together with the accelerator and brake pedals **161**, **163**. As noted above, the steering wheel **171** bears user operable input controls of the cruise control HMI **18** and LSP control HMI **20**. As in the case of a conventional vehicle, the steering wheel **171** has a 'set-speed' control **173**, actuation of which enables a user to activate the cruise control system **16** to maintain the current vehicle speed. The wheel **171** also has a 'LSP' control activation button **172** for activating the LSP control system **12** and a resume button **173R**. The resume button **173R** may be used to control both the 'on-highway' cruise control system **16** when driving on road, and the LSP control system **12** when driving off-road, to resume a previously set (user defined) set-speed.

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If the vehicle is operating on-highway, depression of set-speed control 173 causes the cruise control system 16 to activate provided the current vehicle speed is within the operating range of the cruise control system 16. Depression of the '+' control 174 causes the cruise control system 16 to increase the set-speed whilst depression of the '-' control 175 causes the cruise control system 16 to decrease the set-speed. It will be appreciated that '+' and '-' controls may be on a single button in some arrangements, such as a rocker-type button. In some embodiments, the '+' control 174 may function as a 'set-speed' control, in which case set-speed control 173 may be eliminated.

If the vehicle is operating off-highway, depression of set-speed control 173 causes the LSP control system 12 to activate and operate as described above, provided vehicle speed is within the operating range of the LSP control system 12.

In some embodiments, the system may further comprise a 'cancel' button operable to cancel speed control by the LSP control system 12. In some embodiments, the LSP system may be in either one of an active condition or a standby condition. In the present embodiment the LSP control system 12 is also operable to assume an intermediate condition in which vehicle speed control by the LSP control system 12 is suspended but a hill descent control (HDC) system or the like may remain active if already active. Other arrangements are also useful.

With the LSP control system 12 active, the user may increase or decrease the vehicle set-speed by means of the '+' and '-' buttons 174, 175. In addition, the user may also increase or decrease the vehicle set-speed by lightly pressing the accelerator or brake pedals 161, 163 respectively. In some embodiments, with the LSP control system 12 active the '+' and '-' buttons 174, 175 are disabled. This latter feature may prevent changes in set-speed by accidental pressing of one of these buttons, for example when negotiating difficult terrain where relatively large and frequent changes in steering angle may be required. Other arrangements are also useful.

FIG. 4 shows a flow process to illustrate the interaction between the cruise control system 18 and the LSP control system 12. If cruise control is active when the user tries to activate the LSP control system 12 via the LSP control HMI 20, a signal is sent to the cruise control system 16 to cancel the speed control routine. The LSP control system 12 is then initiated and the vehicle speed is maintained at the target speed selected by the user via the LSP HMI 20. It is also the case that if the LSP control system 12 is active, operation of the cruise control system 16 is inhibited. The two systems 12, 16 therefore operate independently of one another, so that only one can be operable at any one time, depending on the speed at which the vehicle is travelling.

In some embodiments, the cruise control system 16 may hand over vehicle speed control to the LSP control system 12 if a user reduces set-speed of the vehicle 100 to a value within the operating speed range of the LSP control system 12. Similarly, in some embodiments the LSP control system 16 may hand over vehicle speed control to the cruise control system 16 if a user raises vehicle set-speed to a value that is within the operating range of the cruise control system 16. Other arrangements are also useful.

In some embodiments, the cruise control HMI 18 and the LSP control HMI 20 may be configured within the same hardware so that, for example, the speed selection is input via the same hardware, with one or more separate switches being provided to switch between the LSP input and the cruise control input.

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FIG. 5 illustrates the means by which vehicle speed is controlled in the LSP control system 12. As described above, a speed selected by a user (set-speed) is input to the LSP control system 12 via the LSP control HMI 20. A vehicle speed sensor 34 associated with the powertrain 129 (shown in FIG. 1) provides a signal 36 indicative of vehicle speed to the LSP control system 12. The LSP control system 12 includes a comparator 28 which compares the set-speed (also referred to as a 'target speed' 38) selected by the user with the measured speed 36 and provides an output signal 30 indicative of the comparison. The output signal 30 is provided to an evaluator unit 40 of the VCU 10 which interprets the output signal 30 as either a demand for additional torque to be applied to the vehicle wheels 111-115, or for a reduction in torque applied to the vehicle wheels 111-115, depending on whether the vehicle speed needs to be increased or decreased to maintain the speed that has been selected by the user. An increase in torque is generally accomplished by increasing the amount of powertrain torque delivered to a given position of the powertrain, for example an engine output shaft, a wheel or any other suitable location. A decrease in torque to a value that is less positive or more negative may be accomplished by decreasing powertrain torque delivered to a wheel and/or by increasing a braking force on a wheel. It is to be understood that in some embodiments in which a powertrain 129 has an electric machine operable as a generator, negative torque may be applied by the powertrain 129 to one or more wheels. It is to be understood that a brake controller 13 may nevertheless be involved in determining whether brake torque is required to be provided by an electric machine of a powertrain 129, and whether brake torque should be provided by an electric machine or a friction-based foundation braking system 22.

An output 42 from the evaluator unit 40 is provided to the powertrain controller 11 and brake controller 13 which in turn control a net torque applied to the vehicle wheels 111-115. The net torque may be increased or decreased depending on whether there is a positive or negative demand for torque from the evaluator unit 40. Thus, in order to initiate application of the necessary positive or negative torque to the wheels, the evaluator unit 40 may command that additional power is applied to the vehicle wheels and/or that a braking force is applied to the vehicle wheels, either or both of which may be used to implement the change in torque that is necessary to maintain the target vehicle speed. In the illustrated embodiment the torque is applied to the vehicle wheels individually so as to maintain the target vehicle speed, but in another embodiment torque may be applied to the wheels collectively to maintain the target speed. In some embodiments, the powertrain controller 11 may be operable to control an amount of torque applied to one or more wheels by controlling a driveline component such as a rear drive unit, front drive unit, differential or any other suitable component. For example, one or more components of the driveline 130 may include one or more clutches operable to allow an amount of torque applied to one or more wheels to be varied. Other arrangements are also useful.

Where a powertrain 129 includes one or more electric machines, for example one or more propulsion motors and/or generators, the powertrain controller 11 may be operable to modulate torque applied to one or more wheels by means of one or more electric machines. In some embodiments, the one or more electric machines may be operable as either a propulsion motor or a generator under the control of the powertrain controller 11. Thus the powertrain controller 11 may in some embodiments be con-

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trolled to apply more positive or more negative torque to one or more wheels by means of an electric machine.

The LSP control system **12** also receives a signal **48** indicative of a wheel slip event having occurred. This may be the same signal **48** that is supplied to the on-highway cruise control system **16** of the vehicle, and which in the case of the latter triggers an override or inhibit mode of operation in the on-highway cruise control system **16** so that automatic control of the vehicle speed by the on-highway cruise control system **16** is suspended or cancelled. However, the LSP control system **12** is not arranged to cancel or suspend operation in dependence on receipt of a wheel slip signal **48** indicative of wheel slip. Rather, the system **12** is arranged to monitor and subsequently manage wheel slip so as to reduce driver workload. During a slip event, the LSP control system **12** continues to compare the measured vehicle speed with the desired vehicle speed as input by the user, and continues to control automatically the torque applied across the vehicle wheels so as to maintain vehicle speed at the selected value. It is to be understood therefore that the LSP control system **12** is configured differently to the cruise control system **16**, for which a wheel slip event has the effect of overriding the cruise control function so that manual operation of the vehicle must be resumed, or the cruise control function reset.

A further embodiment of the invention (not shown) is one in which the vehicle is provided with a wheel slip signal **48** derived not just from a comparison of wheel speeds, but further refined using sensor data indicative of the vehicle's speed over ground. Such speed over ground determination may be made via global positioning (GPS) data, or via a vehicle mounted radar or laser based system arranged to determine the relative movement of the vehicle and the ground over which it is travelling. A camera system may be employed for determining speed over ground in some embodiments.

At any stage of the LSP control process the user can override the function by depressing the accelerator pedal **161** and/or brake pedal **163** to adjust the vehicle speed in a positive or negative sense. However, in the event that a wheel slip event is detected via signal **48**, the LSP control system **12** remains active and control of vehicle speed by the LSP control system **12** is not suspended. As shown in FIG. 5, this may be implemented by providing a wheel slip event signal **48** to the LSP control system **12** which is then managed by the LSP control system **12**. In the embodiment shown in FIG. 1 the SCS **14** generates the wheel slip event signal **48** and provides it to the LSP control system **12** and cruise control system **16**.

A wheel slip event is triggered when a loss of traction occurs at any one of the vehicle wheels. Wheels and tyres may be more prone to losing traction when travelling on snow, ice or sand and/or on steep gradients or cross-slopes, for example, or in environments where the terrain is more uneven or slippery compared with driving on a highway in normal on-road conditions. Embodiments of the present invention therefore find particular benefit when the vehicle is being driven in an off-road environment, or in conditions in which wheel slip may commonly occur. Manual operation by the user in such conditions can be a difficult and often stressful experience and may result in an uncomfortable ride. Embodiments of the present invention enable continued progress to be made at a relatively low target speed without the need for user intervention.

The vehicle **100** is also provided with additional sensors (not shown) which are representative of a variety of different parameters associated with vehicle motion and status. These may be inertial systems unique to the speed control system

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or part of an occupant restraint system or any other subsystem which may provide data from sensors such as gyros and/or accelerometers that may be indicative of vehicle body movement and may provide a useful input to the LSP control system **12**. The signals from the sensors provide, or are used to calculate, a plurality of driving condition indicators (also referred to as terrain indicators) which are indicative of the nature of the terrain conditions over which the vehicle is travelling. The signals are provided to the VCU **10** which determines the most appropriate control mode for the various subsystems on the basis of the terrain indicators, and automatically controls the subsystems accordingly. This aspect of the invention is described in further detail in our co-pending patent application nos. GB1111288.5, GB1211910.3 and GB1202427.9, the contents of each of which is incorporated herein by reference.

The sensors (not shown) on the vehicle include, but are not limited to, sensors which provide continuous sensor outputs to the VCU **10**, including wheel speed sensors, as mentioned previously and as shown in FIG. 5, an ambient temperature sensor, an atmospheric pressure sensor, tyre pressure sensors, wheel articulation sensors, gyroscopic sensors to detect vehicular yaw, roll and pitch angle and rate, a vehicle speed sensor, a longitudinal acceleration sensor, an engine torque sensor (or engine torque estimator), a steering angle sensor, a steering wheel speed sensor, a gradient sensor (or gradient estimator), a lateral acceleration sensor which may be part of the stability control system (SCS), a brake pedal position sensor, a brake pressure sensor, an accelerator pedal position sensor, longitudinal, lateral and vertical motion sensors, and water detection sensors forming part of a vehicle wading assistance system (not shown). In other embodiments, only a selection of the aforementioned sensors may be used.

The VCU **10** also receives a signal from the steering controller **170C**. The steering controller is in the form of an electronic power assisted steering unit (ePAS unit). The steering controller **170C** provides a signal to the VCU **10** indicative of the steering force being applied to steerable road wheels **111**, **112** of the vehicle **100**. This force corresponds to that applied by a user to the steering wheel **171** in combination with steering force generated by the controller **170C**.

The VCU **10** evaluates the various sensor inputs to determine the probability that each of a plurality of different control modes for the vehicle subsystems is appropriate, with each control mode corresponding to a particular terrain type over which the vehicle is travelling (for example, mud and ruts, sand, grass/gravel/snow). The VCU **10** then selects which of the control modes is most appropriate and controls various vehicle parameters accordingly.

The nature of the terrain over which the vehicle is travelling (as determined by reference to the selected control mode) may also be utilised in the LSP control system **12** to determine an appropriate increase or decrease in drive torque to be applied to the vehicle wheels. For example, if the user selects a target speed that is not suitable for the nature of the terrain over which the vehicle is travelling, the system **12** is operable to automatically adjust the vehicle speed downwards by reducing the speed of the vehicle wheels. In some cases, for example, the user selected speed may not be achievable or appropriate over certain terrain types, particularly in the case of uneven or rough surfaces. If the system **12** selects a set-speed that differs from the user-selected set-speed (i.e. target speed), a visual indication

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of the speed constraint is provided to the user via the LSP HMI **20** to indicate that an alternative speed has been adopted.

As described above, the LSP control system **12** is operable to command a required amount of torque to be applied to one or more driven wheels of the vehicle **100** in order to cause the vehicle to travel at the user-selected set-speed. If whilst the LSP control system **12** is active the driver depresses the accelerator pedal **161** to demand additional powertrain torque above the amount currently demanded by the LSP control system **12**, driver torque demand takes priority and the powertrain **129** is controlled so as to meet driver demand. In the present embodiment, if driver demand exceeds that demanded by the LSP control system **12**, the LSP control system **12** remains active, i.e. the LSP control system **12** continues to calculate an amount of powertrain torque and brake torque that the powertrain controller **11** and brake controller **13** should command be applied to the driven wheels of the vehicle **100** in order to travel at the set-speed. In this way, once a driver releases the accelerator pedal **161**, the LSP control system **12** resumes control of vehicle speed.

It is to be understood that the LSP control system **12** may command the required amount of torque by generating a value of a torque parameter. The powertrain controller **11** may be configured to develop the required amount of torque depending on the value of this torque parameter, which may have a value that corresponds to the required amount of torque without necessarily being equal to the amount of required torque. Thus, for example, the LSP control system **12** may generate a code, such as a number such as 10020, which may correspond to a required powertrain torque of 150 Nm, different codes being generated for different required amounts of powertrain torque. In response to receipt of a command to generate powertrain torque at a level of (say) 10020, the powertrain controller **11** may therefore control the powertrain **11** to generate 150 Nm of torque. Other arrangements are also useful.

As described above, the LSP control system **12** may command the powertrain **129** to develop a required amount of torque at a given location, such as at an output shaft of the engine **121**, at an input shaft of the transmission **124**, an output shaft of the transmission **124**, a wheel or any other suitable location. It is to be understood that if the LSP control system **12** is arranged to control the powertrain **129** to apply a given amount of torque at a location other than a wheel, such as an output shaft of the engine **121**, the torque delivered at a wheel may be calculated based on a gear ratio between the engine output shaft and wheel. The control system **12** may be operable to command the powertrain **129** to establish a given gear ratio between a given position of the powertrain **129** and wheel, so as to establish a desired torque at the wheel. Thus, whilst the LSP control system **12** commands the powertrain to generate a given amount of torque at a given position (and may command the generation of given amounts of torque at a plurality of locations, particularly in a powertrain **129** having a plurality of motors such as an engine and an electric propulsion motor), the LSP control system **12** may also be operable to ensure that the torque delivered to a wheel is a required value by suitable control of the gear ratio. Other arrangements are also useful. In some embodiments, the LSP control system **12** may be provided with data corresponding to a gear ratio between a given position of the powertrain **129** and one or more wheels, and command application of an amount of torque to the given position of the powertrain **129** so as to obtain a required amount of torque at the one or more wheels.

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FIG. 7 illustrates implementation of powertrain torque control in the vehicle **100** of FIG. 1. As can be seen from FIG. 7 an accelerator pedal controller **161C** receives an input signal from accelerator pedal **161**. The signal corresponds to pedal position and in response to this signal the pedal controller **161C** determines the amount of torque that the powertrain **129** should be commanded to develop. Signal pedal_tq_rq corresponding to this amount of torque is then output to a max_pass function block **315**. In some embodiments, powertrain controller **11** may be involved in determining the value of pedal_tq_rq in response to a value of pedal position provided by the pedal controller **161C**.

When the LSP control system **12** is active the LSP control system **12** determines an amount of torque LSP_tq_rq that is to be developed by the powertrain **129**. The value LSP_tq_rq generated by the LSP control system **12** is supplied to a modifier function block **320** that provides an output torque value mod_tq to max_pass function block **315**.

The max_pass function block **315** outputs to a driveability filter **350** a signal max_out corresponding to signal LSP_tq_rq or pedal_tq_rq in dependence on which signal has the higher value. The signal having the higher value is output to the driveability filter **350**. The driveability filter **350** is in the form of a low pass filter and generates an output signal tq_cmd corresponding to the actual amount of torque that is to be developed by the powertrain **129**. The driveability filter is tuned to mitigate the effects of driveline shunt so as to enhance driver enjoyment of the vehicle. In the present embodiment, low pass filtering of powertrain torque demand (being a value of demanded engine output shaft torque in the present embodiment) reduces the rate of change of demanded torque made by the driver or LSP control system thereby reducing driveline shunt.

The driveability filter **350** provides an output tq_cmd to powertrain controller **11** corresponding to the amount of torque to be developed by the powertrain **129** of the vehicle **100**. Driveability filters such as that shown in FIG. 7 are well known, and may be tuned according to the torque damping characteristics of the vehicle powertrain **129**.

The value tq_cmd output by the driveability filter **350** is also input to the modifier function block **320**. The modifier function block **320** is shown in more detail in FIG. 9 and will be discussed further below. In calculating the value of mod_tq, the modifier function block **320** is configured to subtract from the torque request value LSP_tq_rq generated by the LSP control system **12** the difference between the value of LSP_tq_rq and the currently demanded value of powertrain torque (tq_cmd) output by the driveability filter **350**. This feature has the effect of reducing the value of tq_cmd more rapidly. This in turn results in the vehicle **100** slowing to the set-speed more rapidly. The modifier function block **320** may be tuned such that the rate at which vehicle speed slows to the set-speed corresponds more closely to that which would be experienced if the LSP control system **12** were not active and a user of the vehicle **100** released the accelerator pedal **161** in order to slow the vehicle **100** to the set-speed. The amount subtracted from the torque request value LSP_tq_rq may be given by the difference between the value of LSP_tq_rq and current value of tq_cmd multiplied by a factor, for example by a factor 0.5, 2, 3, 4, 5, 10 or any other suitable value. The factor chosen may depend on one or more characteristics of the filter **350**.

It is to be understood that if the torque value applied to the driveability filter **350** were not modified by the modifier function block **320**, the amount of time taken for vehicle speed to fall to the target speed may be significantly longer in some circumstances. The relatively slow response of the

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vehicle 100 in reducing speed, compared with operation without the LSP control system 12, may result in reduced driver confidence in the LSP control system 12.

Embodiments of the present invention have the advantage that, when the LSP control system 12 is active and a driver has intervened to increase powertrain torque above that demanded by the LSP control system 12 by depressing the accelerator pedal 161, the rate at which vehicle speed slows to the target speed may be increased following driver lift-off from the accelerator pedal 161. The rate may be increased to correspond more closely to that which would be experienced if the LSP control system 12 were not functioning, and a driver were to release his or her foot from the accelerator pedal 161 in order to slow the vehicle 100.

It is to be understood that if the vehicle 100 is travelling at a speed of (say) 5 kph over terrain with LSP control system 12 active and in control of vehicle speed, the driver may temporarily increase vehicle speed to (say) 15 kph whilst travelling over a certain portion of the route. When the driver depresses the accelerator pedal 161 to accelerate the vehicle 100, the max_pass function block 315 allows the pedal_tq_rq signal to take priority since it will exceed the LSP_tq_rq signal for a period of time. During this period, the LSP control system 12 recognises that vehicle speed has exceeded the set-speed and responds by gradually reducing the amount of torque demanded by the system 12 in order to slow the vehicle 100.

If the driver releases the accelerator pedal 161, the value of pedal_tq_rq falls to a minimum torque request value. The modifier function block 320 subtracts from the current value LSP_tq_rq generated by the LSP control system 12 the difference between the value of LSP_tq_rq and the currently demanded value of powertrain torque (tq_cmd) output by the driveability filter 350, subject to a minimum allowable value of mod_tq. The minimum allowable value may be a negative value corresponding to powertrain braking, typically due to engine braking and powertrain losses. Thus the value of mod_tq may be less than the value of LSP_tq_rq, and closer to a value corresponding to minimum powertrain torque demand (being the value assumed by signal pedal_tq_rq in the present embodiment). Accordingly, a rate at which the value of tq_cmd approaches the value of LSP_tq_rq may be increased.

As noted above, the powertrain 129 may be capable of developing (and develop in use) negative torque due at least in part to engine over-run (compression) braking. Such braking may be provided by the powertrain 129 due to inertia when the vehicle 100 is moving and a value of commanded torque tq_cmd is sufficiently low.

The minimum amount of powertrain torque may correspond to a prescribed minimum amount of torque that the powertrain can provide under the prevailing conditions. The minimum torque may depend on a temperature of one or more components of the powertrain such as an engine temperature, an electric machine temperature, a gearbox temperature and one or more other components in addition or instead.

By way of illustration of the importance of embodiments of the present invention, FIG. 8(a) illustrates an alternative implementation of powertrain torque control not being an implementation according to an embodiment of the present invention. In the arrangement shown, the two torque demand signals LSP_tq_rq and pedal_tq_rq described above are provided to max_pass function block 315 in a similar manner to the arrangement of FIG. 7(a). Signal LSP_tq_rq is generated by LSP control system 12 when the LSP control system 12 is active, whilst signal pedal_tq_rq is generated

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by accelerator pedal controller 161C in response to driver actuation of accelerator pedal 161. The max_pass function block 315 outputs to a driveability filter 350 a signal max_out corresponding to signal LSP_tq_rq or pedal_tq_rq in dependence on which signal has the higher value. The signal having the higher value is output to the driveability filter 350. As described above with respect to FIG. 7(a), the driveability filter 350 applies a low pass filter to the signal input thereto, generating an output signal tq_cmd1 corresponding to the actual amount of torque that is to be developed by the powertrain 129.

FIG. 8(b) illustrates a manner in which the amount of commanded torque tq_cmd1 varies as a function of time in one example scenario.

At time t0, the LSP control system 12 is active and generating a torque command signal LSP_tq_rq commanding generation by the vehicle powertrain of an amount of torque T1. Signal LSP_tq_rq is input to the max_pass function block 315. Also at time t0, a driver of the vehicle has depressed accelerator pedal 161 to demand torque T2 that is greater than T1. Consequently the accelerator pedal controller 161C outputs to the max_pass function block 315 signal pedal_tq_rq corresponding to a torque value T2. The max_pass function block 315 outputs the higher of these two values (i.e. the value of pedal_tq_rq, T2) to the driveability filter 350.

At time t1, the driver releases the accelerator pedal 161. The value of pedal_tq_rq therefore falls substantially to zero. The max_pass function block 315 detects that signal LSP_tq_rq now has the higher value, and therefore outputs signal LSP_tq_rq to the driveability filter 350. The value of tq_cmd1 generated by the driveability filter 350 therefore begins to fall as a function of time, as shown in FIG. 8(b) until at time t2 the value of tq_cmd1 is substantially equal to LSP_tq_rq.

By way of comparison, the value of tq_cmd that would be generated by driveability filter 350 in the embodiment illustrated in FIG. 7 in substantially the same scenario is also illustrated in FIG. 8(b). It can be seen that the value of tq_cmd falls to the value commanded by the LSP control system 12 more rapidly when the driver releases the accelerator pedal 161 at time t1. Where intervention by the driver has resulted in a substantial increase in vehicle speed, the effect of the more rapid fall in the value of tq_cmd is that the vehicle speed reduces to the set-speed more rapidly than it would in the arrangement of FIG. 8(a).

FIG. 9 illustrates functioning of the modifier function block 320 of FIG. 7. As described above, signals LSP_tq_rq and tq_cmd are input to the function block 320. The signal LSP_tq_rq is applied to a difference function element 320a and to a summing function element 320h. The value of tq_cmd is also input to difference function element 320a, which outputs to a dead zone function element 320b a value corresponding to the difference in torque values between signals LSP_tq_rq and tq_cmd.

The dead zone function element 320b outputs a signal that is substantially equal to the signal input thereto. However, if the signal input thereto is within a prescribed amount of zero (whether positive or negative), the dead zone function element 320b outputs a signal corresponding to substantially zero torque.

The signal output by the dead zone function element 320b is input to a minimum-pass (min-pass) function element 320c and to a maximum-pass (max-pass) function element 320d.

The min-pass function element 320c compares the signal received with a zero-value signal and outputs a signal

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corresponding to the smaller of the two signals. Thus, if the signal input from the dead-zone function element **320b** is negative (corresponding to a value of tq_cmd greater than LSP_tq_rq), the min-pass function element **320c** outputs this signal to a modifier negative function element **320f**.

The max-pass function element **320d** compares the signal received with a zero-value signal and outputs a signal corresponding to the larger of the two signals. Thus, if the signal input from the dead-zone function element is positive (corresponding to a value of tq_cmd less than LSP_tq_rq), the max-pass function element **320d** outputs this signal to a modifier positive function element **320e**.

The modifier negative function element **320f** multiplies the signal input thereto from the min-pass function element **320c** by a prescribed factor. In the embodiment illustrated in FIG. 9 this factor is 5 although other values are also useful. Similarly, the modifier positive function element **320e** multiplies the signal input thereto from the min-pass function element **320d** by a prescribed factor. In the embodiment illustrated in FIG. 9 this factor is 5 although other values are also useful. The factors employed by the modifier function elements **320e**, **320f** may be different in some embodiments.

The purpose of multiplying the signals by a prescribed factor greater than 1 is that the rate at which a speed of the vehicle **100** approaches the target speed value may be increased.

The values output by the modifier positive function element **320e** and modifier negative function element **320f** are input to a summing function element **320g** which outputs a value corresponding to the sum of the signals input thereto. The resulting value is output to summing function element **320h**. Summing function element **320h** outputs a value corresponding to the sum of the signal input from summing function element **320g** and signal LSP_tq_rq .

The signal output by the summing function element **320h** is in turn input to a torque saturation function element **320i**. The torque saturation function element **320i** is arranged to output a signal corresponding to the signal input thereto but placing an upper (positive) limit and a lower (negative) limit on the value of the signal output thereby. For input signal values between the positive and lower limits, the output signal value is substantially equal to the input signal value.

The signal output by the summing function element **320h** is input to the driveability filter **350**.

Embodiments of the present invention provide a control system having speed control functionality that attempts to enhance user enjoyment of vehicle operations. The control system allows a user to over-ride speed control by actuating an accelerator control to request a powertrain to develop torque exceeding the instant value required by the speed control function to achieve or maintain a set-speed.

If the user subsequently releases the accelerator control without cancelling the speed control function, vehicle speed reduces to the current speed employed by the speed control system more quickly than might occur in systems not according to the present invention. The system does this by reducing more rapidly the amount of torque that the powertrain has been commanded to develop towards a value corresponding to that required to achieve the current set-speed.

The more rapid reduction in torque may be achieved by reducing the value of powertrain torque input to a powertrain torque filter (sometimes referred to as a driveability filter, being a filter typically arranged to mitigate the effects of driveline shunt) below the amount required by the speed control function at a given moment in time. This causes an

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output of the filter more quickly to tend towards the value of torque required by the speed control function. Other arrangements are also useful.

It will be understood that the embodiments described above are given by way of example only and are not intended to limit the invention, the scope of which is defined in the appended claims.

Throughout the description and claims of this specification, the words “comprise” and “contain” and variations of the words, for example “comprising” and “comprises”, means “including but not limited to”, and is not intended to (and does not) exclude other moieties, additives, components, integers or steps.

Throughout the description and claims of this specification, the singular encompasses the plural unless the context otherwise requires. In particular, where the indefinite article is used, the specification is to be understood as contemplating plurality as well as singularity, unless the context requires otherwise.

Features, integers, characteristics, compounds, chemical moieties or groups described in conjunction with a particular aspect, embodiment or example of the invention are to be understood to be applicable to any other aspect, embodiment or example described herein unless incompatible therewith.

The invention claimed is:

1. A control system for a vehicle operable to implement a speed control function, the control system being operable to: receive an input of a target speed at which the vehicle is intended to travel;

determine an instantaneous value of a primary target speed torque parameter corresponding to an amount of torque that should be developed at a given position in a powertrain in order to control the vehicle to travel at the target speed; and

filter by means of a filter a value of a torque parameter input thereto to generate a filtered torque parameter value,

the system being operable to command the powertrain to develop at said position an amount of torque corresponding to the filtered torque parameter value,

wherein the system is operable to input to the filter a secondary target speed torque parameter value determined in dependence on the primary target speed torque parameter value and a current value of the filtered torque parameter generated by the filter.

2. A system according to claim 1 operable to determine the value of the secondary target speed torque parameter in dependence on a difference between the value of the primary target speed torque parameter and the current value of the filtered torque parameter.

3. A system according to claim 1 operable wherein if the value of the primary target speed torque parameter is less than a current value of the filtered torque parameter, the value of the secondary target speed torque parameter is set to a value that is less than the primary target speed torque parameter by an amount that is determined in dependence on the primary target speed torque parameter value and the current value of filtered torque parameter.

4. A system according to claim 3 operable to input to the filter a value of secondary target speed torque parameter that is less than the primary target speed torque parameter by an amount corresponding to the difference between the filtered torque parameter value and the primary target speed torque parameter value.

5. A system according to claim 3 operable to input to the filter a value of secondary target speed torque parameter that is less than the primary target speed torque parameter value

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by an amount substantially equal to the difference between the filtered torque parameter value and the primary target speed torque parameter value, multiplied by a factor.

6. A system according to claim 1 operable such that if the value of primary target speed torque parameter is greater than that of the filtered torque parameter the value of secondary target speed torque parameter input to the filter is increased to a value above that of the primary target speed torque parameter value, the amount of the increase being determined in dependence on the primary target speed torque parameter value and the current value of filtered torque parameter.

7. A system according to claim 6 operable to input to the filter a value of secondary target speed torque parameter that is greater than the primary target speed torque parameter value by an amount substantially equal to the difference between the filtered torque parameter value and the primary target speed torque parameter value.

8. A system according to claim 6 operable to input to the filter a value of secondary target speed torque parameter that is greater than the primary target speed torque parameter value by an amount substantially equal to the difference between the filtered torque parameter value and the primary target speed torque parameter value, multiplied by a factor.

9. A system according to claim 1 operable to limit an upper and lower value of secondary target speed torque parameter input to the filter to respective upper and lower saturation values.

10. A system according to claim 1 operable to input to the filter the higher of the secondary target speed torque param-

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eter value and a value of a driver demanded torque parameter corresponding to a setting of a user-operable accelerator control.

11. A system according to claim 1 wherein the filter is arranged to subject the value of torque parameter input thereto to a low pass filter.

12. A system according to claim 1 operable to receive a user input of the target speed.

13. A vehicle comprising a control system according to claim 1.

14. A method of controlling a speed of a vehicle comprising carrying out the following steps using at least one controller on the vehicle: receiving an input of a target speed at which the vehicle is intended to travel; determining an instantaneous value of a primary target speed torque parameter corresponding to an amount of torque that should be developed at a given position in a powertrain in order to control the vehicle to travel at the target speed; filtering a value of a torque parameter input thereto to generate a filtered torque parameter value; and commanding the powertrain to develop at said position an amount of torque corresponding to the filtered torque parameter value, wherein filtering the value of torque parameter input thereto comprises filtering a secondary target speed torque parameter value determined in dependence on the primary target speed torque parameter value and a current value of the filtered torque parameter.

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